



قال تعالى

سُلْنَا بِنَالْبَيِّنَاتِ وَأَنْزَلْنَا مَعَهُمُ
الْكِتَابَ وَالْمِيزَانَ فِيهِ الْبُلْكُ الْمُبِينَ
وَأَنْزَلْنَا مَقْدِيرًا وَمَنَافِعً لِلنَّاسِ وَلِيَعْلَمَ
اللَّهُمَّ مَن يَنْصُرُهُ وَرَسُلُهُ بِالْغَيْبِ إِنَّ اللَّهَ قَوِيٌّ
عَزِيزٌ .

سورة الحديد - الآية {25}



إلي وطني العزيز الغالي السـودان
دمـترمزاً للسيادة عالياً بين الدول

إلي جامعتي العريقة جامعة السـودان
دمـتي صرحاً شامخاً ومنارة للعلم

إلي أستاذتي ومعلمي الأجلاء الأكـارم
دمـتم شراعاً مرفوع ومشاعل للعلم

إلي زملائي وأخوانني من دفعـة 2006

دمـتم شعاراً للوفاء والإخـاء

إلي أهلي ورفاق دربي وأصدقائي الأعزـاء

دمـتم دعماً ودرعاً لي في وجه الزـمن

إلي أبائنا الأكارم العـظـماء

دمـتم رايـلت من التضـحيـات ورمـزاً للعطـاء

ولـكي وحدـك بـعد الله أـنـحـني تقـسـياً وـتـجـيلاً

أـمـمي أـدـامـك الله نـبـضاً لـروحـي

إـلـي كلـ هـوـلـاء وـلـكـلـ منـ كـانـ لهـ بـصـمةـ فـيـ حـيـاتـنـاـ نـهـيـ هـذـاـ الـبـحـثـ ، وـنـأـمـلـ مـنـ اللهـ أـنـ يـحـقـقـ الـفـائـدـةـ الـعـلـمـيـةـ الـمـرـجـوـةـ مـنـهـ

Acknowledgement

First of all, giving thanks to Allah,
And a special thanks to supervisor

Dr . Al i Mohammed Hamdan Adam.

I can't forget giving thanks to everyone who helped me and gave me a new hope for successful.

I am gratefully acknowledges the financial support of the lectures and all the staff of Sudan University's, School of Mechanical Engineering for all what they offered throughout my educational path in the university.

At the last “thanks” for every one stood beside me until the last dot.

Abstract

A finite-time thermodynamic modelling and simulation of irreversible Joule-Braytoncycle has been developed taking into account the variabilityof specific heats for working fluid due to temperature variation.

The effect of parameters on the engine was discussed, which is the internal irreversibility.

A program was developed by using MATLAB software to perform the necessary calculations ofthermodynamic model.

According to the results obtained, it was found that the results obtained from the thermodynamic model compared with the results obtained from experiments are convergent, and may be used in actual engine designs and applications.

تجريـد

تم تطوير نموذج محاكاة لمحرك يعمل وفقاً لدورة جول - برايتون اللاعکوسية وذلك باستخدام طريقة الحركيات الحرارية محدودة الزمن مع الأخذ بعين الاعتبار التغير في قيم السعات الحرارية بالنسبة للمادة العاملة تبعاً للتغير في قيم درجات الحرارة. تم مناقشة عوامل اللاعکوسية الداخلية التي تؤثر على المحرك.

تم استخدام برنامج ماتلاب لإجراء الحسابات اللازمة للنموذج. وفقاً للنتائج التي تم الحصول عليها تبيّن أن النتائج التي تم الحصول عليها من هذا النموذج مقارنة مع النتائج التي يتم الحصول عليها من التجارب المعملية متقاربة وبالتالي يمكن استخدام هذه النتائج مستقبلاً في تصميم المحرك وتطبيقاته المختلفة.

Table of contents

الاستهلال -----		I
الإهداء -----		II
Acknowledgement -----		III
Abstract -----		IV
Abbreviations & Symbols-----		VIII
List of tables -----		XI
List of figures -----		XII

1. <i>Introduction</i>		
1.1	Research overview -----	2
1.2	Research problem -----	3
1.3	Research importance -----	3
1.4	Research objectives -----	3
1.5	Research approach -----	3
1.6	Methodology -----	3
1.7	Anticipated Results -----	4
1.8	Thesis layout -----	4
 2. <i>Literature Review</i>		
2.1	Introduction -----	6
2.2	Historical background-----	6
2.3	Literature review-----	7
 3. <i>Theoretical Framework</i>		
3.1	Equilibrium thermodynamics-----	16
3.2	First law efficiency -----	16
3.3	Second law efficiency-----	17
3.4	Non-equilibrium theory-----	18
3.5	Reversibility and irreversibility -----	19
3.6	Irreversibilities -----	20
3.7	Endoreversible thermodynamics -----	24
3.8	Endoreversible system (Novicov engine) -----	24
 4. <i>Thermodynamic Modeling</i>		
4.1	Introduction -----	29
4.2	Joule-Brayton engine thermodynamic cycle model -----	29
4.3	Thermodynamic model analysis -----	30
4.4	Pre calculations -----	38
4.5	Solving methodology-----	40
4.6	Program flow chart-----	43

5. Results and Discussion		
5.1	Introduction -----	46
5.2	Thermodynamic model input data details -----	46
5.3	Effect of internal irreversibility -----	46
6. Conclusion and Recommendations		
6.1	Conclusion -----	56
6.2	Recommendations -----	57
References -----	58	
Appendix: Program code		

Abbreviations & Symbols

A/Abbreviations:

C.V	Calorific value
L.M.T.	Logarithmic mean temperature
S.F.C.	Specific fuel consumption

B/ List of symbols:

Symbol	Function	Unit
$\frac{A}{F}$	Air-fuel ratio	
C_p	The constant pressure specific heat	kJ/kg.K
C_{p_a}	The constant pressure specific heat for air	kJ/kg.K
$C_{p_{mix}}$	The constant pressure specific heat for mixture	kJ/kg.K
C_v	The constant volume specific heat	kJ/kg.K
D	The diffusion coefficient	m^2/s
η_I	First law efficiency	%
η_{II}	Second law efficiency	%
η_C	Isentropic efficiency for compressor	%
η_T	Isentropic efficiency for turbine	%
η_{th}	Thermal efficiency	%
η_{Carnot}	Carnot cycle efficiency	%
η_{Nov}	Novicov efficiency	%
I	The flux of electric current	$N.m^2/C$
J	The diffusion flux	$kmol/m^2.s$
K_H	Finite heat conductance for source reservoir	$kW/m.K$
K_L	Finite heat conductance for sink reservoir	$kW/m.K$
λ	The thermal conductivity	$kW/m.K$
M_i	The molecular weight for component i	$kg/kmol$
\dot{m}_a	The mass flow rate of air	kg/s
\dot{m}_f	The mass flow rate of fuel	kg/s
n_a	Number of moles for air	$kmol$
\dot{W}_{net}	The net output power	kW
\dot{Q}_{add}	The added heat flow-rate	kW
R	The gas constant	kJ/kg.K

R_u	The universal gas constant	kJ/kg.K
r_P	The pressure ratio	
<i>S. F. C.</i>	Specific fuel consumption	kg/kW.hr
T_1	Temperature of inlet air	K
T_{iH}	Internal high temperature for source reservoir	K
T_{iL}	Internal low temperature for sink reservoir	K
T_H	Temperature of source thermal reservoir	K
T_L	Temperature of sink thermal reservoir	K
x_i	The mass fraction for component i .	
y_i	The mole fraction for component i .	

List of Tables

Table 4.1(a): Coefficients for species temperature-dependent specific heats [$T \leq 1000 K$] -----	32
Table 4.1(b): Coefficients for species temperature-dependent specific heats [$1000 < T < 3200 K$] -----	32
Table 4.2: Molecular weight of air constituent gases -----	33
Table 4.3: Volumetric and gravimetric analysis for combustion products	40
Table 5.1: The technical and thermodynamic engine specifications -----	46

List of Figures

Figure 3.1 – Friction between piston and cylinder walls cause irreversible process -----	21
Figure 3.2 – Unrestrained expansion -----	22
Figure 3.3 – Heat transfer through a finite temperature difference -----	23
Figure 3.4 – Model of the endoreversible Novicov engine -----	25
Figure 4.1 – T-s diagram of an air standard Joule - Brayton cycle model	29
Figure 5.1 – Effect of internal irreversibility on compressor power (\dot{W}_C)-----	47
Figure 5.2 – Effect of internal irreversibility on turbine power (\dot{W}_T)-----	48
Figure 5.3 – Effect of internal irreversibility on net power (\dot{W}_{net})-----	49
Figure 5.4 – Effect of internal irreversibility on specific fuel consumption (S. F. C.)-----	50
Figure 5.5 – Effect of internal irreversibility on actual efficiency (η_{Act}).-	51
Figure 5.6 – Effect of internal irreversibility on three type of efficiencies ($\eta_{Act}, \eta_{Carnot}, \eta_{Nov}$) for compressor and turbine values is 0.7 -----	52
Figure 5.7 – Effect of internal irreversibility on three type of efficiencies ($\eta_{Act}, \eta_{Carnot}, \eta_{Nov}$) for compressor and turbine values is 0.8-----	53
Figure 5.8 – Effect of internal irreversibility on three type of efficiencies ($\eta_{Act}, \eta_{Carnot}, \eta_{Nov}$) for compressor and turbine values is 0.9-----	54